

Experimental Model Identification for Flexible Multibody Mechanisms Through the Flexible Natural Coordinate Formulation and Vision-Based Measurements

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## Experimental model identification for flexible multibody mechanisms through the flexible natural coordinate formulation and vision-based measurements

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## Abstract

This work presents a novel framework for the experimental model identification of flexible multibody mechanisms. It is shown that by exploiting the flexible natural coordinate formulation (FNCF) and vision-based measurements, it becomes possible to use a least-squares model identification method without the need for time-expensive model simulations in between optimizer iterations.

By using the FNCF multibody formulation [4], the equations of motion for a flexible multibody mechanism are given by the formula as shown in Eq. (1):

$$\begin{cases} \boldsymbol{M}\ddot{\boldsymbol{q}} + \boldsymbol{K}\boldsymbol{q} + \boldsymbol{\Xi}\left(\begin{bmatrix}\boldsymbol{1}\\\boldsymbol{q}\end{bmatrix} \otimes \boldsymbol{\lambda}\right) = \boldsymbol{B}\left(\begin{bmatrix}\boldsymbol{1}\\\boldsymbol{q}\end{bmatrix} \otimes \boldsymbol{f}\right) \\ \boldsymbol{\phi}\left(\begin{bmatrix}\boldsymbol{1}\\\boldsymbol{q}\end{bmatrix} \otimes \begin{bmatrix}\boldsymbol{1}\\\boldsymbol{q}\end{bmatrix}\right) = \boldsymbol{0} \end{cases}$$
(1)

Here,  $\boldsymbol{M}$  and  $\boldsymbol{K}$  are the mass and stiffness matrices respectively,  $\boldsymbol{q}$  is the vector of generalised coordinates,  $\boldsymbol{\lambda}$  is the vector of Lagrangian multipliers,  $\boldsymbol{f}$  is the vector of external excitation forces,  $\boldsymbol{\Xi}$  and  $\boldsymbol{B}$  are constant projection matrices and  $\boldsymbol{\phi}$  is the collection of constraint equations. In case of the FNCF formulations, the vector of generalized coordinates  $\boldsymbol{q} = \left[ \boldsymbol{p}^{\top}, \boldsymbol{r}^{\top}, \boldsymbol{\gamma}^{\top}, \boldsymbol{\delta}^{\top} \right]^{\top}$  consists of the rigid body translations  $\boldsymbol{p}$  and rotations  $\boldsymbol{r}$ , the flexibility participation factors  $\boldsymbol{\delta}$  and a special variable  $\boldsymbol{\gamma} = \boldsymbol{\delta} \otimes \boldsymbol{r}$ . This variable  $\boldsymbol{\gamma}$  is unique for the FNCF formulation and ensures that both the mass matrix  $\boldsymbol{M}$  and the stiffness matrix  $\boldsymbol{K}$  are constant. This property opens up the possibility for a least-squares model identification as shown in Eq. (2):

$$\underset{\boldsymbol{M},\boldsymbol{K},\boldsymbol{\lambda}}{\operatorname{arg\,min}} \left\| \boldsymbol{M} \ddot{\boldsymbol{q}} + \boldsymbol{K} \boldsymbol{q} + \boldsymbol{\Xi} \left( \begin{bmatrix} 1 \\ \boldsymbol{q} \end{bmatrix} \otimes \boldsymbol{\lambda} \right) - \boldsymbol{B} \left( \begin{bmatrix} 1 \\ \boldsymbol{q} \end{bmatrix} \otimes \boldsymbol{f} \right) \right\|$$
(2)

Where, instead of the often time-consuming model simulations in between optimizer iterations, simple matrix multiplications are used. However, this requires the knowledge of the full vector of generalized coordinates  $\boldsymbol{q}$  and the external excitation  $\boldsymbol{f}$  for each time step. By only using conventional sensors (e.g., accelerometers, strain gauges, encoders), this is difficult to achieve especially for the case of flexible multibody mechanisms where a measurement of both the rigid body motion (i.e.,  $\boldsymbol{p}$  and  $\boldsymbol{r}$ ) and the flexible deformations (i.e.,  $\boldsymbol{\delta}$ ) are required. Therefore this research proposes to use vision-based measurements as they can provide full-field motion measurements of the mechanism with a sufficient accuracy and spatial density in order to extract individual component deformations [5]. The vision-based motion tracking in this research uses an affine Lucas-Kanade optical flow [1] in combination with a Procrustes motion separation [3] in order to obtain the components rigid body motions ( $\boldsymbol{p}$  and  $\boldsymbol{r}$ ) and deformation motions. Both a hybrid modal decomposition [2] and an singular value decompositions are exploited in order to decompose the deformation motion into individual modes and participation factors  $\boldsymbol{\delta}$ .

As a validation platform, a planar slider-crank mechanism is used as shown in Fig. 1. Here, the crank is considered as a rigid component while the connecting rod is assumed to be flexible. The setup is recorded with a Photron SA-Z High Speed Camera at 40000 frames per second while the crank was



Figure 1: The planar slider-crank mechanism used as validation platform.

rotating at a speed of 60 rad/s. The external excitation f is obtained by a torque readout from the driving servomotor. Fig. 2 shows the extracted first deformation mode of the connecting rod using the hybrid modal decomposition method.



Figure 2: First deformation mode of the connecting rod (x-deformation: deformation along the horizontal connecting rod axis, y-deformation: deformation along the vertical connecting rod axis).

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