



Artificial Finger Consisting of Closed Linkages and Single Planetary Gear System: an Approach to Grasp Stabilizing with Distal Hyperextension Joint

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Artificial finger consisting of closed linkages and Single Planetary Gear System: an approach to grasp stabilizing with a distal hyperextension joint

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Abstract

The authors have developed a five-finger robot hand [1-4]. Figure. 1 shows the structure used in four fingers (index, mid, medicinal, and little). It has two serially connected closed four-bar linkages and a Single Planetary Gear System (SPGS). It is a two-degree-of-freedom (2-DOF) mechanical system that drives actively and passively. It constitutes a variable stiffness mechanism that achieves sensorless shape-fitting motion (envelope grasping) when grasping an object of unknown shape. This mechanism gains robustness on harsh environmental usages since no electric or electronic devices are embedded in the finger part. However, the mechanism of such an under-actuation system requires stability in its motion, specifically, stable object grasping or pinching. It must be achieved on a mechanical basis, not by control maneuver.

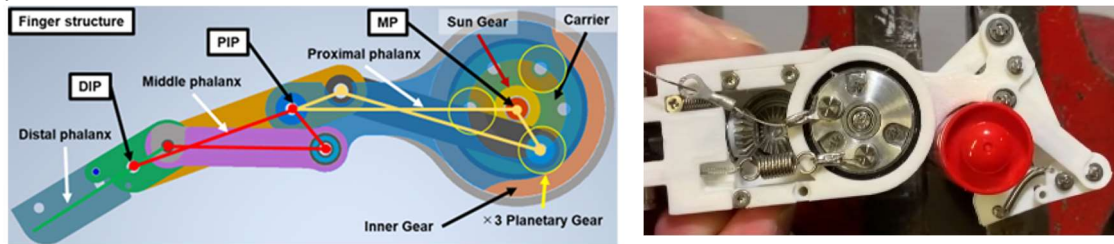


Figure 2: Finger mechanism

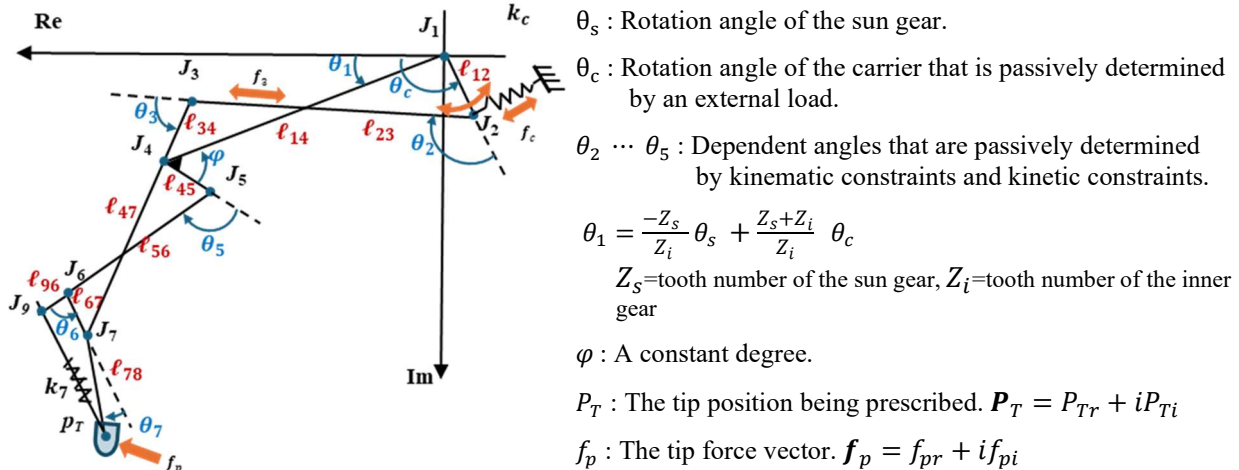


Figure 1: Kinematic representation of the finger having a Hyperextension joint.

Recently, we developed a new finger mechanism with a Hyperextension joint at the fingertip aiming to stabilize the pinching motion passively. This joint has a linear compression spring. This abstract just shows the mathematical formulation of the finger mechanism to compute its static motion and force that the fingertip applies to an external object. The analysis will be reported in the presentation.

Figure .2 shows the kinematic representation of the finger mechanism. We determine unknown 9 variables $\theta_c, \theta_2, \theta_3, \theta_5, \theta_6, \theta_7, f_3, f_{pr}, f_{pi}$ under solar gear rotation θ_s being given and the tip position $P_{Tr} + iP_{Ti}$ being prescribed.

$$\begin{aligned} J_1 &= \{0, 0\}, J_2 = l_{12}e^{i\theta_c}, J_3 = J_2 + l_{23}e^{i(\theta_c+\theta_2)}, J_4 = l_{14}e^{i\theta_1} = J_3 + l_{34}e^{i(\theta_c+\theta_2+\theta_3)} \\ J_5 &= J_4 + l_{45}e^{i(\theta_1+\pi-\varphi)}, J_6 = J_5 + l_{56}e^{i(\theta_1+\pi-\varphi+\theta_5)}, \\ J_7 &= J_4 + l_{47}e^{i(\theta_c+\theta_2+\theta_3)} = J_6 + l_{67}e^{i(\theta_1+\pi-\varphi+\theta_5+\theta_6)} \\ P_T &= J_6 + l_{78}e^{i(\theta_1+\pi-\varphi+\theta_5+\theta_6+\theta_7)}, J_9 = J_6 + l_{96}e^{i(\theta_1+\pi-\varphi+\theta_5)} \end{aligned} \quad (1)$$

First, we derive the kinematic constraints. Two joints J_4 and J_7 have two passes to reach their positions due to closed linkages. It gives us the first to fourth constraint equation.

$$\begin{aligned} c_1 + ic_2 &\equiv \ell_1 e^{i\theta_1} - (J_3 + \ell_4 e^{i(\theta_c + \theta_2 + \theta_3)}) = 0 \\ c_3 + ic_4 &\equiv (J_3 + (\ell_4 + \ell_5) e^{i(\theta_c + \theta_2 + \theta_3)}) - (J_6 + e^{i(\theta_1 + \pi - \varphi + \theta_5 + \theta_6)}) = 0 \end{aligned} \quad (2)$$

Second, we derive the kinetic constraints. Let f_3 be the internal force vector acting on the link $J_2 - J_1$. Then, f_3 is expressed by $f_3 = pf_3(J_3 - J_2)/\ell_3$, where pf_3 is the force value. The sum of the force vectors acting at the J_2 is in parallel to the vector $J_2 - J_1$ on an equilibrium state, which provides a constraint described by outer product of $f_3 + f_c$ and $J_2 - J_1$ being zero.

$$c_5 = (f_3 + f_c) \times (J_2 - J_1) \quad (3)$$

, where, the operator “ \times ” is defined as $\mathbf{p} \times \mathbf{q} = p_r q_i - p_i q_r$ for $\mathbf{p} = p_r + ip_i$ and $\mathbf{q} = q_r + iq_i$.

Let f_7 be the force vector acting on the spring link $J_c - J_2$. Then, f_7 is expressed by $f_7 = pf_7(J_c - J_2)/\ell_{c2}$ ($\ell_{c2} = \text{spring link length}$), where pf_7 is the force value. The sum of the force vectors acting at the J_7 is in parallel to the vector $J_7 - P_T$ on an equilibrium state, which provides a constraint described by outer product of $f_7 + f_p$ and $J_7 - P_T$ being zero.

$$c_6 = (f_7 + f_p) \times (J_7 - P_T) \quad (4)$$

The fact that the torque acting at J_4 is zero provides us another constraint,

$$c_7 = f_3 \times (J_4 - J_3) + (f_7 + f_p) \times (J_4 - P_T) \quad (5)$$

A prescribed tip position $\hat{P}_T = \hat{P}_{Tr} + i\hat{P}_{Ti}$ gives us the following two constraint equations,

$$c_8 + ic_9 = \hat{P}_{Tr} + i\hat{P}_{Ti} - (J_6 + \ell_{78} e^{i(\theta_1 + \pi - \varphi + \theta_5 + \theta_6 + \theta_7)}) = 0 \quad (6)$$

The set of infinitesimal variation of nine constraint equations ;

$\Delta c = (\Delta c_1 \Delta c_2 \Delta c_3 \Delta c_4 \Delta c_5 \Delta c_6 \Delta c_7 \Delta c_8 \Delta c_9)$, should also hold zero vector during motion ($\Delta c = \mathbf{0}_9$). In addition, we find solutions of nine variables ($\theta_c, \theta_2, \theta_3, \theta_5, \theta_6, \theta_7, f_3, f_{pr}, f_{pi}$) by repeatedly executing the Newton-Raphson method under nine constraint equations being sustained.

$$\varphi_1 \rightarrow \theta_s, \varphi_2 \rightarrow \theta_c, \varphi_3 \rightarrow \theta_1, \varphi_4 \rightarrow \theta_2, \varphi_5 \rightarrow \theta_3, \varphi_6 \rightarrow \theta_4, \varphi_7 \rightarrow f_{pr}, \varphi_8 \rightarrow f_{pi} \quad (10)$$

$$\Delta c = \frac{\partial c}{\partial \varphi} \Delta \varphi + \frac{\partial c}{\partial \theta_s} \Delta \theta_s = \begin{pmatrix} \partial c_1 / \partial \varphi_1 & \cdots & \partial c_1 / \partial \varphi_9 \\ \vdots & \ddots & \vdots \\ \partial c_9 / \partial \varphi_1 & \cdots & \partial c_9 / \partial \varphi_9 \end{pmatrix} \Delta \varphi + \frac{\partial c}{\partial \theta_s} \Delta \theta_s = \Lambda \Delta \varphi + \alpha \Delta \theta_s = 0 \quad (11)$$

$$\Delta \varphi = -\Lambda^{-1} \alpha \Delta \theta_s \quad (12)$$

$$\varphi_k = \varphi_{k-1} + \Delta \varphi, k = 1, 2, 3 \dots \quad (13)$$

$$\varphi_{k,j} = \varphi_{k,j-1} - \Lambda^{-1} c, j = 1, 2, \dots \quad (14)$$

where, k is an iteration number. The numerical errors generated by the linear approximation will be compensated by employing the Newton-Raphson convergency procedure in the cycle.

We analyze the finger motion and the reaction force of the fingertip in this way. At this point, we analyze how the force produced at the fingertips varies due to sun gear rotation input and how the hyperextension of the fingertips affects on. This abstract shows a new finger structure and the mathematical basis of the analysis. We will show the simulation results at the time of presentation.

References

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